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# **Research on Aerodynamic Axial Flow Fan Efficiency Improvement**

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**Abstract**: Fans used to move air in commercial and industrial buildings account for a significant portion of the total energy consumption. Since much of the city's planned structures have yet to be built, and since both the cost of electricity and its availability are on the rise, there is an immediate need for fan designs that minimise energy consumption. This research compared two fan designs in terms of the overall effort needed to achieve certain output levels using a retrofit experimental approach. Due to their equal output characteristics, an aerodynamic axial flow fan and a belt-driven centrifugal fan are compared to find out which one is better. The aerodynamically optimised design of the axial flow fan used in this research drastically reduces the total power input, connected load, and rated motor KW. An additional advantage of this fan type is the smaller air handling unit. The current centrifugal fan design's performance characteristics are evaluated and documented. These stringent requirements prompted the development of a state-of-the-art axial flow fan with an aerodynamic design. After the retrofitting procedure, the performance of the improved fan design is assessed by a second comprehensive performance analysis. Considering the mechanical efficiency of the fan, the system's efficiency is calculated both before and after the refit. By using computational fluid dynamics (CFD), we were able to simulate the two systems' operation and investigate the impact of various design parameters on efficiency. These parameters included the twist of the rotor blades, the geometry of the hub, and the guiding vanes. The Ansys flow solver allows us to perform the simulation and determine the power requirements of the fan in order to achieve the desired airflow. Using a return on investment (ROI) technique, we may determine whether it is feasible to replace the current fan with a new, more efficient one. Thanks to this cutting-edge, environmentally conscious method, HVAC energy consumption may be reduced by up to 30%. Computational fluid dynamics study of the system may help optimise the design further for better performance and efficiency. Aerofoil guiding, retrofitting, aerodynamic axial flow, and computational fluid dynamics performance parameters are some of the topics discussed.

### Introduction

The circulation of air is fundamental to all living things. Fan is the mechanical device that was created to meet this need. In commercial and industrial air movement systems, fans account for over 40% of the total power consumption. Better energy-efficient fan designs might cut fan power consumption by as much as half. According to comprehensive research, the typical efficiency of industrial fans range from 33% to 55%. Fan inefficiency is mostly caused by the design of the fan's impeller or rotor. The overall mechanical efficiency of the fan has the potential to reach 92% with this optimisation. There is a pressing need for energy-efficient fans since a large portion of the city's new buildings are still under construction, and because power prices are rising and supplies are becoming more scarce. The rated motor kW, connected load, and total power input of the axial flow fan used in this research may be significantly decreased with the aid of its optimised aerodynamic design. The smaller air handling unit is an extra perk of this fan type. The amount of force needed to pump the specified air volume is dictated by the drag that is produced by the blade design. Reduced drag from an aerofoil's design means less thrust is required to pump large volumes, which means less power air consumption from the engine.



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set up to turn the fan's impeller. In order to improve the fan's efficiency, we looked at a number of aerodynamic design characteristics. Industry standards are used to realistically quantify the enhanced performance. We will submit the revised design to industry experts for execution so that we may transition to renewable energy sources to power these machines, which will have a good effect



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on the environment.

#### TypesOfFans

Considerations like as efficiency, space constraints, material type, flow rate, pressure, and volume dictate the choice of fan and blower. Different kinds of fan different efficiency. designs have Centrifugal flow and axial flow are the two main types of fan motion. When air enters and exits a centrifugal flow device, the direction of the airflow may be either curved. backward forward curved. inclined, or radial. Axial flow (also known as propeller, tube, or vane axial) occurs when air flows into and out of a fan in a straight line.

#### Figure1.1.TypesOfFans

a) CentrifugalFan:Types The major types of centrifugal fan are: radial, forward curved and backward curvedRadial fans are industrial workhorses because of their high static pressures (upto 1400mm WC) and ability to handle heavily contaminated airstreams. Because of their simpledesign, radial fans are well suited for high temperatures and medium blade speeds.Forwardtip curved fans are used in clean environments and operate at lower temperatures. They are well suited for low tip speed and high-airflow work - they are best suited formoving large volumes of air against relatively pressures. low Backwardinclined fans aremore efficient forward-curved than fans. Backward-inclined fans reach their peakpower consumption and then power demand drops off well within the reusableairflow range. Backward-inclined fans are

known as "non-overloading" becausechangesinstaticpressure donotoverloadthemotor.

b) AxialFlowFan:Types The major types of axial flow fans are: tube a Tube axial fan has a wheel inside acylindrical housing, with close clearance between blade and housing to improveair flowefficiency. The wheel turns faster than propeller fans, enabling operationunder high65%. Vane axial fans are similar to tube axial, but with addition of guide vanes thatimproveefficiencybydirectin gandstraighteningtheflow.Asare sult, they have a higher static pressure with less dependence on the used generally for



andmoderatetemperatures.Th eyexperiencealargechangeina irflowwithsmallchangesin static pressure. They handle large volumes of air at low pressure or free delivery.Propeller fans are often used indoors as exhaust fans. Outdoor applications include aircooledcondensersandcoolingt owers.Efficiencyislowapprox imately50%orless.



Figure1.2.TypesOf AxialFlowFans

#### 1.2 Typesofbladedesigns





Radial blades: Forming a rotor which is essentially a large paddle wheel, this designresults in a relatively inefficient fan with power consumption higher than that usingthe much more common backward inclined blade. Its inherent mechanical strengthandresistancetowearme anitisgenerallyusedwhenhighqu antitiesofabrasivedustarepresen tinthegasstream, or when very hig hgastemperatures are expected.

Forward curved : A forward curved centrifugal fan is characterized by its cylindricalshape and lots of small blades the on circumference of the impeller. In the exampleshownbelow,thefanrot atesinaclockwisedirection.

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Backwardcurved

Anefficientands trongshape,thisdesignisacos t-effectivealternativetothe backward curved design but with flat plate blades instead of curved. This resultsinaslightlylowereffici ency,compensatedbytheeasi erfittingofliner

CentrifugalFans			AxialFlowFans		
Туре	Characteristics	TypicalApp lications	Туре	Characteristics	TypicalApp lications
Radial	High Pressure,Mediu m Flow.Efficienc yclosetotube axialfans. Powerincreases continuously	VariousIndustrial applications,suita ble fordust laden, moist air /gases	Propeller	LowPressure,Hig h flow,Loweff iciency,Peakeffici encyclosetopoint of free airdelivery (zero staticpressure)	Aircirculation Ventilation,ex haust.
ForwardCurv edBlades	MediumPressure, highflow ,dipinpressure curve,eff iciencyhigherthanra dialfans,power risescont inuously	LowpressureHV AC, packaged unitssuitableforcl ean and dustladenair/gase s	Tube- axial	MediumPressure, highflow highereff iciencythanpropel lertype,dip in pressure- flowcurve beforepe ak pressure point.	HVAC, dryingove ns,exhaust systems
Backwardcur vedblades	Highpressure, highfl ow, highefficiency, powerreduces as flow increase beyond poin to fhighest efficiency	HVAC, variousindustriala pplications,forced draftfa ns,etc.	Vane- axial	High pressure,mediumf low,dip in pressure– flowcurve,useofg uide vanesim provesefficiency exhausts.	Highpressurea pplicationsincl udingHVAC systems
Airfoilty pe	Same as backward type,hig hestefficiency	Same as backwardcurvedb utforclean airap			

#### ${\tt Table 1. Types Of Fans, Characteristics And Applications}$



#### MethodologyofDesignStudy

centrifugal The fan. however, is more difficult, particularly when high efficiency isrequired. The blades are given an aerofoil to decrease shape the resistance created for theflow of air. Guide vanes are also made in aerofoil shape and the profile is given such thatthe leading edge of the aerofoil first comes into contact of the airflow to take the most of the benefit from the aerofoil design, Apart from the aerofoil design the rotor blades aretwisted along the length which is unique design optimization done in this fan design, thebenefitof whichisdiscussedindetailfurt her.

#### 2.1 Designcriteriaofafan:-

The key factors of design criteria of a fan are , specifications of the fan, selection for theapplication, impellerstressi ng, critical speed of the rotor, fie ldInstallation and Fansound.

#### Table2.SpecificationsofFan

Volumetricflow,m <sup>3</sup> /s	11.20
StaticPressure,Pa	640
Density,Kg/m <sup>3</sup>	1.20
Speed,rpm	1450
Fandiameter,m	0.9

#### 2.2 FactorsAidingInIncreasedEfficiencyOfFan

The contact of the air with the first physical element of the fan itself starts the relationbetween the performance of the fan directly resulting the design criteria. The inlet conehasaveryvasteffectonthe soundlevelsofthefan.Ifasmoo thtransitionwiththehelpofinle t cone is not provided then the air cuts across the

diameter of the fan and high soundlevels are experienced. Hence having an inlet cone helps in reducing fan sound levels.Similarly the hub blades and guide vanes along with the tip clearance between the bladesandtherotorcasinghave influenceonthefanperformanc e.



Figure 2.1 Aerodynamics of high efficient Axial flow fan



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ThetermAxialflowfanindicatesth attheairflowsthroughthefaninana pproximatelyaxial direction. On th einletside, as the flow approachest hefanblades.thedirectionoftheflo wisaxial, in other words, parallel tot heaxisofrotation, provided there ar enoinletvanesorotherrestrictions aheadofthefanwheel.Thefanblad ethendeflectstheairflow.Apropell erisamechanismdesignedtoprodu ceatractiveforceorpush, when sub mergedinafluidmedium.Theprop ellersareaerodynamicelementsth atarecomposedofahuborcentralc oreandanumberofblades.Theoper atingprincipleofaxialflowfansissimplydeflectionofairf low.Pasttheblade,therefore,thepa tternofthedeflectedairflowisofhel icalshape,likeaspiralstaircase.Th isistrueforallthreetypesofaxialflowfans:propellerfans,tubeaxial fans, and vaneaxial fans.

**2.3 TheAerofoildesignoftheblade** Anaerofoilisastreamlineshape.Itsma inapplicationisasthecrosssectionofa nairplanewing. Another application is as the cross section of a fan blade. There are symmetric andasymmetric aerofoils. The aerofoils used in fan blade are asymmetric. Fig.2.2 shows anasymmetric aerofoil that has been developed by the National Advisory Committee foraeronautics (NACA), it normally produces positive pressure on the lower surface of theaerofoilandnegativepressureorsu ctionontheuppersurface. The suction pressureonthetop surface are about twice as large as the positive pressures on the lower surface, but allthese positive and negative pressures push and pull in approximately the same directionandreinforceeachother. The combinationofthesepositiveandnega tivepressures results in a force F. This f orceFcanberesolvedintotwocompon ents:aliftforceL,perpendicular to the relative air velocity; and a drag force D, parallel to the relative airvelocity. Fig.2.3 shows the forces acting on an aerofoil. The lift force L is the usefulcomponent(R:WASETNACA )



#### Figure 2.2. Shape of Asymmetric Aerofoil

Inthefigure2.2theleadingedgeist hepointatthefrontoftheaerofoilth athasmaximumcurvature(minim umradius).Thetrailingedgeisdefi nedsimilarlyasthepointofmaxim umcurvatureattherearoftheaerof oil.Thechordlineisthestraightlin econnecting leading and trailing edges. The chord length, or simply chord, is the length ofthechordline.Thatisthereferen cedimensionoftheaerofoilsectio n.Camberlinearethepoints halfway between chord and upper blade surface. Angle of attack is the anglebetweendirectionof airflowandthechord.

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Figure 2.3. Forces Acting On Aerofoil

The lift that an aerofoil generates depends on the density of the air, the velocity of theairflow, the viscosity and compressibility of the air, the surface area of the aerofoil, theshapeoftheaerofoil, and the ang leoftheaerofoil'sangleofattack.H owever,dependenceon the aerofoil's shape, the angle of attack, air viscosity and compressibility are where Listhelifting force, pisthedensity of ai r,vistherelativevelocityoftheairflow, s is the area of the aerofoil as viewed from an overhead perspective, and CL is the liftcoefficient.

As with lift, the drag of an aerofoil depends on the density of the air, the velocity of theairflow, the viscosity and compressibility of the air, the surface area of the aerofoil, theshape of the aerofoil, and the verycomplex. Thus, they are characterized by a single variable in the lift equation, called thelift coefficient. Due to the complexities of the lift coefficient, it is generally found viaexperimentationinawindtunn elwheretheremaining variablescanbecontrolled.Theref ore,theliftequationisgivenby

#### $\rho v^2 s C L$

angle of attack. The associated complexities with drag andthe aerofoil's shape, angle of attack, the air's viscosity, and air's compressibility are simplified in the drag equation by use of the drag coefficient. The drag coefficient isgenerallyfoundthroughtestingi nawindtunnel, where the drag canb emeasured, and the drag coefficien tiscalculatedbyrearrangingthedr agequation.

# $D = (1/2)\rho v^2 A C D$

 $In the drag equation, \textbf{D} is the drag force, \rho is the density of the air, v is the velocity of the air, A is a reference area, and CD is the drag coefficient.$ 



#### 2.4 Twistedaerofoilblade

The blade experiences a relative wind velocity that is a vector sum of the actual windspeed (V) and the speed caused by the rotation of the rotor ( $\Omega$  r where  $\Omega$  is the rotationalspeed and r is the distance from the rotation axis). In order to produce lift, an aerofoilshape must be oriented so that its rounded leading edge is facing approximately into theairflow direction. But, the airflow direction for a wind turbine is actually a vector sum of the winditselfandtherelativewindcausedbytherotationofthebladethroughtheair.



Figure2.4.Mechanicsofatwistedbladesetup



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Figure 2.5. Mechanics of a twisted bladesetup

Figure 2.4 shows the example of a wind mill blade.TSR=  $\Omega R/V$ , where  $\Omega$ is the angularvelocityoftherotor,Ris thedistancebetweentheaxisofr otationandthetipoftheblade,a nd V is the wind speed.Since the speed of a rotating blade varies from the center to thetip, the angle with which the airflow encounters the aerofoil varies along the blade

(seeFigure2.5).Toaccountfort his,therotorbladesmustbetwis ted

It is important to note that the leading edge of the blade aerofoil mounted on the impelleris facing approximately into the airflow direction across the length of the blade. It is thisdesign criteria which drastically help in reducing the resistance to the

movement of impeller.

#### 2.5 RoundedHub

#### Arounded

hubbenefitisshowninthefigur e above, usually axial flowfansaregivenflathubsorn otasmuchradialshapeasneede d,onesuchhubcanbeshownint heFigure2.6.Before hitting the blades the air comes into contact with the hub, which experiences thefluid and transfers the flow to the blades, if the hub isn't correctly rounded as in ourdesignitwillhavealossin pressureandefficienciesofthef an as the surface of the bladenearthe hub doesn't have good contact to the laminar fluid hence disrupting the smoothflowofairreducingthea bilityoffantocreatehighpressu resandefficiencies.



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Figure 2.6. Rounded Hubandin let cone



Figure 2.7. Comparison between conventional and our Aerodynamic fandesign

#### 2.6 TheGuidevanesinfandesign

The guide vanes at the outlet of air are provided to provide a near laminar effect to thefluid exiting the impeller, this will help to reduce the turbulence and also increases the efficiency of the fanasithelptore duce the losses due to turbulence.

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Turbulence not only causes increased noise levels but also causes massive pressure losses in the moving air. The design incorporates guiding vanes to make the system more efficient and to transfer the impeller's pressure without any loss. Figure 2.8 shows the asymmetrical form of the guiding vanes, which are aerofoils. The fixed guiding vanes' aerofoil design greatly enhanced the system's efficiency. These vanes are used to transfer the impeller-generated pressure without any loss of pressure. 2.7 Adjusting the Guide Vanes' Profiles to Face the Leading Edge Into the Airflow

When air is released from the impeller, it meets the leading edge of the guiding vanes because the aerofoil's rounded leading edge is angled perfectly such that it faces towards the direction of the airflow,



as seen in the picture below.



#### Figure 2.9. In line leading and trailing edge of blade and guidevane

The second optimised part is the small space between the blade tips, which results in less noise and improved efficiency. The rounded tip and inlet cone work together to improve efficiency by distributing airflow more uniformly. As seen in the image below, the airflow is directed to the beginning point of the blade by the rounded hub without any loss, restriction, or diversion. The following are examples of standard axial flow fan designs for your reference.



Figure 2.10. Close upview of bladetip clear ance in our design

This study's aerodynamic fan design stands out due to the narrow space between the housing and the tip of the moving blade. Minimal blade tip clearance reduces noise and improves efficiency. As shown up above, it's so thin that a business card might squeeze through.

 $\label{eq:comparison} 3.1 Comparison between design of conventional axial fan and a erodynamic axial flow fan and a erodynamic axial flow fan a fan arbitrary fan arbitr$ 











 $\label{eq:Figure3.1.Clusteroff} Figure 3.1. Cluster of fandes igns howing conventional designs and aerodynamic fandes ign showing conventional design showing convention$ 

How an axial fan simulator works at its most basic level. The three primary parts of these procedures—computational domain, meshing, and solving—will serve as the framework for this discussion.

An entrance, a casing that contains the geometry of the blades, and an outlet make up the geometric domain. The produced mesh is very sensitive to the actual dimensions and geometries of the three domain components. The fan's performance is affected by its most crucial aerodynamic component, the blade. Being the sole moving portion of the fan, the blade cuts through air and propels it forward. As a first step, we used reverse engineering to model the fan using all of the existing aerofoil drawings and the real design concepts.

Figure 3.2 below shows the 3D CAD models compared to the original design.



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Original Fan Model	3D CAD Model for Analysis
Aerofoil shape of the blade	P
Meroroli snape or the blade	
Twist of the blade along the axis of rotation	



 $\label{eq:Figure3.2.Tableofpictures comparing original model and CAD model$ 



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Figure 3.3. An Isometric view of the Fan 3DCAD model for Analysis

# **CONCLUSIONS:**

An efficient fan design with an aerofoilshaped impeller and guide vanes was investigated through experimental modelling, reverse engineering, and validation using the ANSYS CFX solver. The model was created in Solidworks. In comparison to a traditional centrifugal the aerodynamically built fan fan, significantly reduced power consumption, leading to a 68.75% savings and an increase in fan efficiency from 58.8% to 89.8%. Computer assisted design (CAD) and computer aided engineering (CAE) have helped model and simulate circumstances identical to those in the real experiment, and the findings obtained are within acceptable limits.

Experimental results showed that the new fan design had a number of advantages over the old centrifugal fan design, one of which was a much lower pressure need. Engineers will be able to choose the best fan model for future retrofit projects if this pressure analysis can be done in the simulation programme.

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